

Q.1 In a power plant water is pumped from 80 kPa to 3 MPa. Isentropic efficiency of pump is 0.80. Temperature is kept-constant. Find the specific work (kJ/kg) input for the pump.

- (a) 0.34 (b) 2.48
 (c) 2.92 (d) 3.43

Ans. (d)

Work input for compressor (theoretical)

$$= -V\Delta P$$

$$= -\frac{1}{\rho}(P_2 - P_1)$$

$$= -\frac{1}{1000}(3 \times 10^3 - 80)$$

$$= -2.92 \text{ kJ/kg}$$

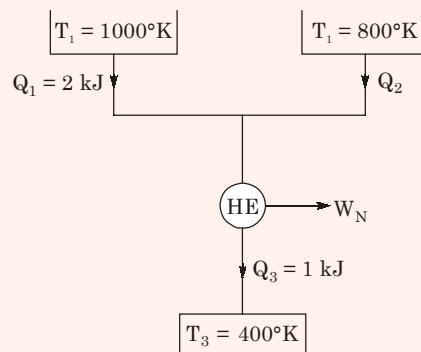
$$\text{Actual work input} = \frac{2.92}{\eta_c} = \frac{2.92}{0.85}$$

$$= 3.43 \text{ kJ/kg}$$

• • • End of Solution

Q.2 A reversible heat engine receive 2 kJ of heat from reservoir at 1000°K and certain amount of heat from another reservoir at 800°K. It rejects 1 kJ of heat to reservoir at 400°K. Find net work output

Solution:



For reversible heat engine

$$(\Delta S)_{\text{Reversible cycle}} = 0$$

$$\Rightarrow -\frac{Q_1}{T_1} - \frac{Q_2}{T_2} + \frac{Q_3}{T_3} = 0$$

$$\Rightarrow -\frac{2}{1000} - \frac{Q_2}{800} - \frac{1}{400} = 0$$

$$\Rightarrow Q_2 = 0.4 \text{ kJ}$$

$$\text{From heat balance, } Q_1 + Q_2 = W_N + Q_3$$

$$\Rightarrow 2 + 0.4 = W_N + 1$$

$$\Rightarrow W_N = 1.4 \text{ kJ}$$

 ● ● ● **End of Solution**

Q.3 For a fully developed flow of water in a pipe of dia. = 10 cm, $V = 0.1 \text{ m/sec}$. Kinematic viscosity = $10^{-5} \text{ m}^2/\text{sec}$. Find Darcy friction factor

Solution:

$$R_e = \frac{VD}{\nu} = \frac{0.1 \times 0.1}{10^{-5}} = 1000$$

Darcy friction factor,

$$f = \frac{64}{R_e} = \frac{64}{1000}$$

$$= 0.064$$

 ● ● ● **End of Solution**

Q.4 For completely submerged body with centre of gravity G, Centre of buoyancy B. Submerged body will be stable if

(a) G above B

(b) G below B

(c) G coincident with B

(d) Independent of G and B

Ans. (b)

B should be above G.

 ● ● ● **End of Solution**

Q.5 Water flow through pipe, whose inner dia. = 10 mm at the rate of 36 kg/hr at 25°C. Viscosity at 25°C = 0.001 kg/m-s. Find Reynold's No.

Solution:

$$d = 10 \text{ mm} = 0.01 \text{ m}$$

$$m = 36 \text{ kg/hr} = \frac{36}{3600} = \frac{1}{100} \text{ kg/sec}$$

$$= 10^{-2} \text{ kg/sec}$$

$$\mu = 0.001 \text{ kg/m-s}$$

$$R_e = ?$$

$$R_e = \frac{\rho V d}{\mu} = \frac{\rho d}{\mu} \times \frac{Q}{A} \quad (Q = \text{flow rate } m^3/\text{sec})$$

$$= \frac{\rho Q d}{\mu \times \frac{\pi}{4} d^2} = \frac{4\rho Q}{\pi d \mu}$$

$$\Rightarrow R_e = \frac{4m}{\pi d \mu} = \frac{4 \times 10^{-2}}{\pi \times 0.01 \times 0.001}$$

$$= 1273.2$$

● ● ● End of Solution

- Q.6** 1.5 kg of water in saturated liquid state at 2 bar ($v_f = 0.00106$, $u_f = 504$ kJ/kg, $h_f = 505$ kJ/kg). Heat added at constant pressure till temperature becomes 400°C ($v = 1.5$, $u = 2967$, $h = 3277$ kJ/kg). Find the heat added.

Solution:

From the First law of thermodynamics

$$dq = du + Pdv$$

$$= (u_2 - u_1) + P(v_2 - v_1)$$

$$= (2967 - 504) + 2 \times 10^2(1.5 - 0.00106)$$

$$= 2762.78 \text{ kJ/kg}$$

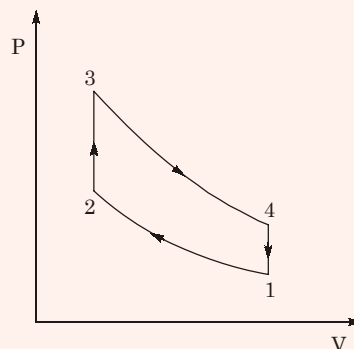
For 1.5 kg of water

$$Q = 2762.78 \times 1.5$$

$$= 4144.182 \text{ kJ}$$

● ● ● End of Solution

- Q.7** For an Otto cycle, given, pressure at inlet = 0.1 MPa, temperature at inlet = 308°K , $\gamma = 1.4$, $R = 288.8$ J/kgK. Compression ratio = 8. Maximum temperature = 2660°K . Find the heat supplied.

Solution:


Given:

$$\begin{aligned}
 P_1 &= 0.1 \text{ MPa} \\
 T_1 &= 308^\circ\text{K} \\
 \gamma &= 1.4 \\
 R &= 288.8 \text{ J/kgK} \\
 r &= 8 \\
 T_3 &= 2660^\circ\text{K} \\
 Q_S &= mC_v(T_3 - T_2) \quad \dots(i) \\
 C_v &= \frac{R}{\gamma - 1} = \frac{288.8}{0.4} \\
 &= 722 \text{ J/kgK} \\
 \frac{T_2}{T_1} &= \left(\frac{V_1}{V_2}\right)^{\gamma-1} = (8)^{0.4} \\
 \Rightarrow T_2 &= 308 \times (8)^{0.4} \\
 &= 707.6^\circ\text{K} \\
 Q_S &= 1 \times 722 (2660 - 707.6) \\
 &= \mathbf{1409.6 \text{ kJ/kg}}
 \end{aligned}$$

 ● ● ● **End of Solution**

Q.8 Given x is random variable, $P(x)$ is probability density

x	1	2	3
$P(x)$	0.3	0.6	0.1

Find standard deviation.

- (a) 0.18 (b) 0.36
 (c) 0.54 (d) 0.6

Ans. (d)

Mean,

$$\begin{aligned}
 \bar{x} &= \frac{\sum f_i x_i}{\sum f_i} \\
 &= \frac{1 \times 0.3 + 2 \times 0.6 + 3 \times 0.1}{0.3 + 0.6 + 0.1} = 1.8
 \end{aligned}$$

Standard deviation

$$\begin{aligned}
 \sigma_x &= \sqrt{\frac{\sum f_i x_i^2}{N} - \bar{x}^2} \\
 &= \sqrt{\frac{0.3 \times 1^2 + 0.6 \times 2^2 + 0.1 \times 3^2}{1} - 1.8^2} \\
 &= \mathbf{0.6}
 \end{aligned}$$

 ● ● ● **End of Solution**

Q.9 $y = f(x)$ is the solution of $\frac{d^2y}{dx^2} = 0$. Boundary conditions are, $y = 5$, $x = 10$.

$$\frac{dy}{dx} = 2 \text{ at } x = 10. \text{ Find } f(15) = ?$$

Solution:

$$\frac{d^2y}{dx^2} = 0$$

$$\Rightarrow \frac{dy}{dx} = C_1$$

$$\text{At } x = 10, \frac{dy}{dx} = 2$$

$$\text{So, } C_1 = 2$$

$$\text{Hence, } \frac{dy}{dx} = 2$$

$$\Rightarrow y = 2x + C_2$$

$$\text{At } x = 10, y = 5$$

$$\text{So } 5 = 2 \times 10 + C_2$$

$$\Rightarrow C_2 = -15$$

$$\text{Hence, } y = 2x - 15$$

$$\begin{aligned} f(15) &= y|_{x=15} \\ &= 2 \times 15 - 15 \\ &= 15 \end{aligned}$$

● ● ● **End of Solution**

Q.10 Value of $\lim_{x \rightarrow 0} \frac{x - \sin x}{1 - \cos x}$ is

(a) 0

(b) 1

(c) 3

(d) Not defined

Ans. (a)

$$\lim_{x \rightarrow 0} \frac{x - \sin x}{1 - \cos x} \quad (0/0 \text{ form, applying 'L' Hospital rule})$$

$$= \lim_{x \rightarrow 0} \frac{\frac{dx}{dx}(x - \sin x)}{\frac{d}{dx}(1 - \cos x)}$$

$$= \lim_{x \rightarrow 0} \frac{1 - \cos x}{0 + \sin x} \quad (0/0 \text{ form})$$

$$\begin{aligned}
 &= \lim_{x \rightarrow 0} \frac{\sin x}{\cos x} \\
 &= 0
 \end{aligned}$$

 ● ● ● **End of Solution**

Q.11 The argument of the complex no. $\frac{1+i}{1-i}$, where $i = \sqrt{-1}$ is

Solution:

$$\begin{aligned}
 \frac{1+i}{1-i} \times \frac{1+i}{1+i} &= \frac{(1+i)^2}{1-i^2} = \frac{1+i^2+2i}{1+1} \\
 &= i = 0 + i
 \end{aligned}$$

$$\begin{aligned}
 \text{Argument } \theta &= \tan^{-1}\left(\frac{y}{x}\right) \\
 &= \tan^{-1}\left(\frac{1}{0}\right) \\
 &= \tan^{-1}(\infty) \\
 &= \frac{\pi}{2}
 \end{aligned}$$

 ● ● ● **End of Solution**

Q.12 The state of stress of a point is given by $\sigma_x = -6$ MPa, $\sigma_y = 4$ MPa and $\tau_{xy} = -8$ MPa. The maximum tensile stress (in MPa) at that point is

Solution:

$$\begin{aligned}
 \sigma_1 &= \frac{\sigma_x + \sigma_y}{2} + \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} \\
 &= \frac{-6 + 4}{2} + \sqrt{\left(\frac{-6 - 4}{2}\right)^2 + (-8)^2} \\
 &= -1 + \sqrt{25 + 64} \\
 &= 8.4339 \text{ MPa}
 \end{aligned}$$

 ● ● ● **End of Solution**

Q.13 For a job in manufacturing process, arrival rate is 5 per shift of 8 hrs following Poisson's distribution. Service rate for the job is 40 min. Find the ideal time (in hr) of the job

(a) $\frac{1}{4}$

(b) $\frac{7}{5}$

(c) $\frac{14}{3}$

(d) $\frac{2}{3}$

Ans. (d)

Arrival rate, $\lambda = \frac{5}{8}$ jobs per hour

Service rate, $\mu = \frac{60}{40} = \frac{3}{2}$ jobs per hour

Fraction of time job is idle

$$= 1 - \frac{\lambda}{\mu} = 1 - \frac{5}{8} \times \frac{2}{3} = \frac{7}{12}$$

 \therefore Idle time = Expected waiting time in the system \times Probability of idleness

$$= \frac{1}{\mu - \lambda} \times \frac{7}{12} = \frac{1}{\frac{3}{2} - \frac{5}{8}} \times \frac{7}{12} = \frac{1}{\frac{12-5}{8}} \times \frac{7}{12}$$

$$= \frac{1}{\frac{7}{8} \times \frac{12}{7}} = \frac{8}{12} = \frac{2}{3}$$

 ● ● ● **End of Solution**
Q.14 Water jet strikes a stationary vertical plate with a volume flow rate of 0.05 m³/sec and exerts a force of 1000 N on the plate. Find out the dia. of jet
Solution:

Given, $Q = 0.05$ m³/sec

$F_N = 1000$ N

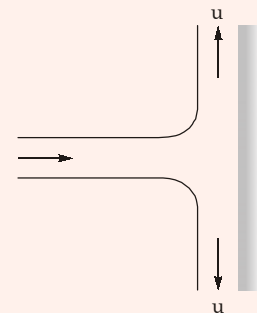
$d = ?$

$F_n = \rho AV^2$

$$= \rho A \times \left(\frac{Q}{A} \right)^2$$

$$= \frac{\rho Q^2}{A} = \frac{4\rho Q^2}{\pi d^2}$$

$$\Rightarrow 1000 = \frac{4 \times 1000 \times 0.05^2}{\pi d^2}$$



$$\Rightarrow \quad d = 0.0564 \text{ m} \\ = 56.4 \text{ mm}$$

 ● ● ● **End of Solution**

Q.15 Which is a CFC refrigerant

- (a) R744 (b) R290
 (c) R502 (d) R718

Ans. (c)

- R744 – CO₂
 R290 – C₃H₈ (Propane)
 R502 – CHClF₃ + CClF₂CF₃
 R718 – Water

 ● ● ● **End of Solution**

Q.16 Given $\begin{pmatrix} 1 & 3 & 0 \\ 2 & 6 & 4 \\ -1 & 0 & 2 \end{pmatrix} = -12$. Find determinant of $\begin{pmatrix} 2 & 6 & 0 \\ 4 & 12 & 8 \\ -2 & 0 & 4 \end{pmatrix} = ?$

Solution:

$$\begin{pmatrix} 2 & 6 & 0 \\ 4 & 12 & 8 \\ -2 & 0 & 4 \end{pmatrix} = 2^n \times -12 \\ = 2^3 \times -12 \\ = -96$$

 ● ● ● **End of Solution**

Q.17 Why it is difficult to weld Aluminium

- (a) low MP of Al
 (b) High thermal conductivity
 (c) Softness
 (d) Specific heat capacity is low

Ans. (a)

 ● ● ● **End of Solution**

Q.18 A pair of spur gear with module 5 mm and a centre distance of 450 mm is used for a speed reduction of 5 : 1. No. of teeth on pinion is

Solution:

Given, $m = 5$
Centre distance (n) = 450 mm
Speed reduction = 5 : 1

i.e., $\frac{Z_G}{Z_P} = \frac{5}{1}$

$\Rightarrow Z_G = 5Z_P \dots(i)$
 $x = r_1 + r_2$

$= \frac{d_1 + d_2}{2}$

$= \frac{mZ_g + mZ_P}{2}$

$\Rightarrow 450 = \frac{5 \times Z_G + 5 \times Z_P}{2}$

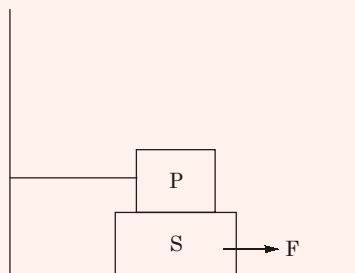
$\Rightarrow Z_G + Z_P = 180$

$\Rightarrow 5Z_P + Z_P = 180$ (from eq. (i))

$\Rightarrow Z_P = 30$

• • • End of Solution

Q.19 Mass P is attached with an inextensible string as shown in figure. Mass of P = 100 kg. mass of S = 150 kg. μ for all surfaces = 0.4. Find F (in kN) required for movement of S.



(a) 0.69

(b) 0.88

(c) 0.98

(d) 1.37

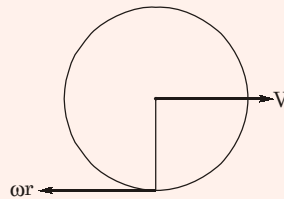
Ans. (d)

$$\begin{aligned} F &= \mu R_1 + \mu R_2 \\ F &= \mu(W_S + W_P) + \mu W_P \\ &= [0.4(150 + 100) + 0.4 \times 100] \times 9.81 \\ &= 1373.4 \text{ N} \\ &= 1.37 \text{ kN} \end{aligned}$$

• • • End of Solution

- Q.20** A wheel is rolling without slipping on a plane surface with the centre velocity of V . What will be velocity of the point of contact
- Velocity $V \perp$ to the surface
 - V in the direction of motion
 - V in the opposite direction of motion
 - Zero

Ans. (d)



Velocity of point of contact = 0.

• • • End of Solution

- Q.21** A rod of length 250 mm is fixed in between two immovable plates. It is heated upto 250°C . Given coefficient of thermal expansion, $\alpha = 1 \times 10^{-5}/^\circ\text{C}$, and $E = 200$ GPa. Stress developed (in MPa) in the beam is

Solution:

Given,

$$\begin{aligned}
 l &= 250 \text{ mm} \\
 \Delta t &= 250^\circ\text{C} \\
 \alpha &= 1 \times 10^{-5}/^\circ\text{C} \\
 E &= 200 \text{ GPa} \\
 \sigma &= ? \\
 \sigma &= E \cdot \alpha \cdot \Delta t \\
 &= 200 \times 10^3 \times 1 \times 10^{-5} \times 250 \\
 &= 500 \text{ MPa}
 \end{aligned}$$

• • • End of Solution

- Q.22** If Taylor's tool life exponent 'n' is 0.2 and the tool change time is 1.5 minute. Then the tool life (in minute) for maximum production rate

Solution:

Given,

$$n = 0.2$$

Tool change time, $T_C = 1.5$ minute

Tool life = ? (For max. production rate)

$$\text{Tool life} = T_{MP} = \left\{ \left(\frac{1}{n-1} \right) T_C \right\}$$

$$= \left(\frac{1}{0.2} - 1 \right) \times 1.5$$

$$= 6 \text{ minute}$$

● ● ● End of Solution

- Q.23** Given initial length ' L_0 ' subjected to drawing process.
 $L(t) = L_0(1 + t^2)$, (t is in minute)
 Find true strain (ϵ_T) in $\text{min}^{-1} = ?$, at the end of '1' minute.

Solution:

$$L(t) = L_0(1 + t^2)$$

$$\epsilon_T = \int d\epsilon = \int \frac{dL}{L}$$

$$L = L_0(1 + t^2)$$

$$\Rightarrow dL = L_0 \times 2t \, dt$$

$$\epsilon_T = \int_0^1 \frac{2tL_0 dt}{L_0(1+t^2)} = \int_0^1 \frac{2t dt}{(1+t^2)}$$

$$\text{Now, Let } (1 + t^2) = z$$

$$\Rightarrow 2t \, dt = dz$$

$$\text{at } t = 0, z = 1$$

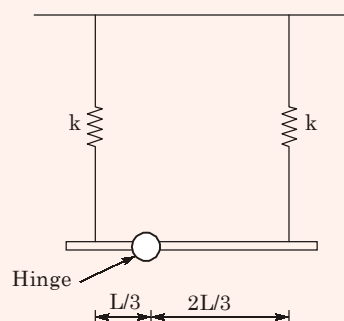
$$\text{at } t = 1, z = 2$$

$$\therefore \epsilon_T = \int_1^2 \frac{dz}{z} = \log_e (z) \Big|_1^2$$

$$= \log_e 2 - \log_e 1$$

$$= 0.693 \text{ min.}^{-1}$$

● ● ● End of Solution

Q.24


Mass of the beam is ' m ' and spring stiffness is ' k '. A hinge is attached with the beam as shown above. What will be the natural frequency of the system.

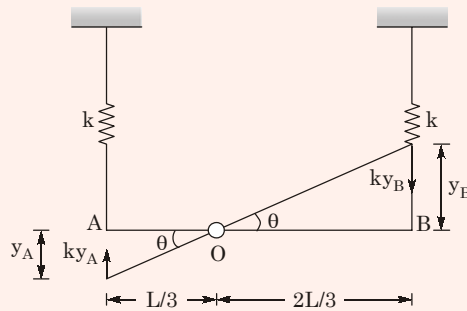
(a) $\sqrt{\frac{k}{m}}$

(b) $\sqrt{\frac{k}{2m}}$

(c) $\sqrt{\frac{2k}{m}}$

(d) $\sqrt{\frac{5k}{m}}$

Ans. (d)



$$\theta = \frac{y_A}{(L/3)} = \frac{y_B}{(2L/3)}$$

$$I_0 = I_c + m \left(\frac{2L}{3} - \frac{L}{2} \right)^2$$

$$I_0 = \frac{mL^2}{12} + \frac{mL^2}{36}$$

$$I_0 = \frac{mL^2}{9}$$

Taking $\Sigma M_0 = 0$

$$\Rightarrow \left(ky_A \times \frac{L}{3} \right) + \left(ky_B \times \frac{2L}{3} \right) + I_0 \ddot{\theta} = 0$$

$$\Rightarrow k\theta \left(\frac{L}{3} \right)^2 + k\theta \left(\frac{2L}{3} \right)^2 + \frac{mL^2}{9} \ddot{\theta} = 0$$

$$\Rightarrow \left(\frac{mL^2}{9} \right) \ddot{\theta} + \left(\frac{5kL^2}{9} \right) \theta = 0$$

$$\Rightarrow \ddot{\theta} + \frac{5k}{m} \theta = 0$$

$$\Rightarrow \ddot{\theta} + \omega_n^2 \theta = 0$$

$$\therefore \omega_n = \sqrt{\frac{5k}{m}}$$

• • • End of Solution

- Q.25** Which one of the following is odd one out
- (a) WEKO (b) IQWA
(c) FNTX (d) NVBD

Ans. (d)

• • • **End of Solution**

- Q.26** Consider the following:
- I. Mating gear is a higher pair
II. Revolute pair is lower pair
- (a) Both are correct (b) I is correct, II is incorrect
(c) I is incorrect, II is correct (d) Both are incorrect

Ans. (a)

• • • **End of Solution**

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